

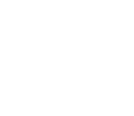
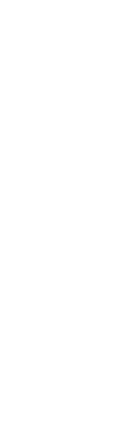
# SCE 594: Special Topics in Intelligent Automation & Robotics

Lecture 12: Rigid Body Dynamics II



# Outline

- Recap last lectures
- In depth analysis



# Outline

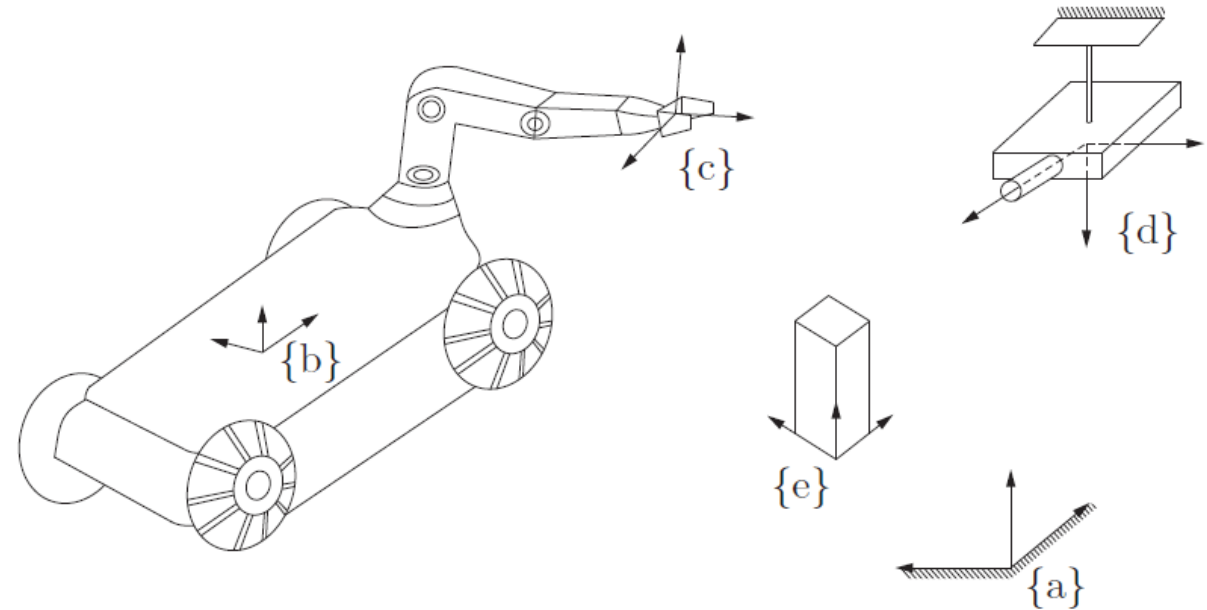
- Recap last lectures
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# Recap: Kinematic Modeling Notation

- A frame will be denoted by  $\Psi_i$  or  $\{i\}$ .
- The relative pose of  $\{i\}$  with respect to  $\{k\}$  is described by

$$H_i^k = \begin{pmatrix} R_i^k & \xi_i^k \\ 0 & 1 \end{pmatrix} \in SE(3), \quad R_i^k \in SO(3), \quad \xi_i^k \in \mathbb{R}^3$$



# Recap: Kinematic Relations

## 1. Point mass translation :

- Configuration:

$$\xi_b^s \in \mathbb{R}^3$$

- Rate-of-change of configuration:

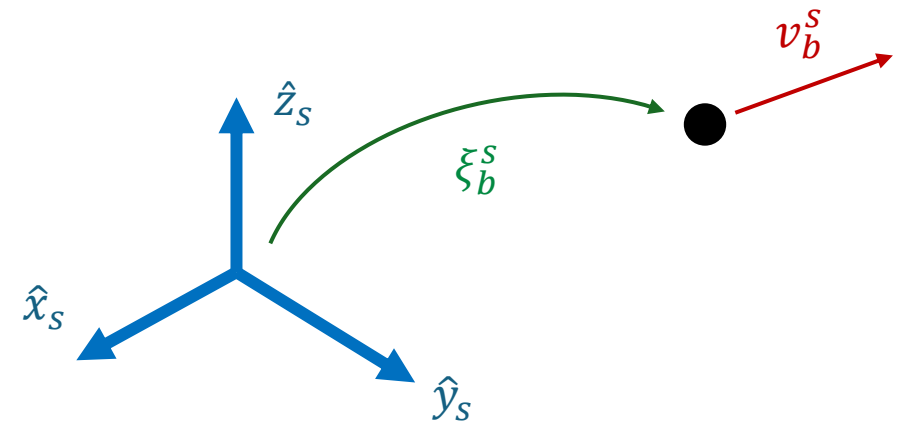
$$\dot{\xi}_b^s \in \mathbb{R}^3$$

- Velocity expressed in  $\{s\}$ :

$$v_b^{s,s} \in \mathbb{R}^3$$

- Kinematic relation:

$$\dot{\xi}_b^s = v_b^{s,s}$$



# Recap: Kinematic Relations

## 2. Rigid body rotation :

- Configuration:

$$R_b^S \in SO(3)$$

- Rate-of-change of configuration:

$$\dot{R}_b^S \in T_{R_b^S} SO(3)$$

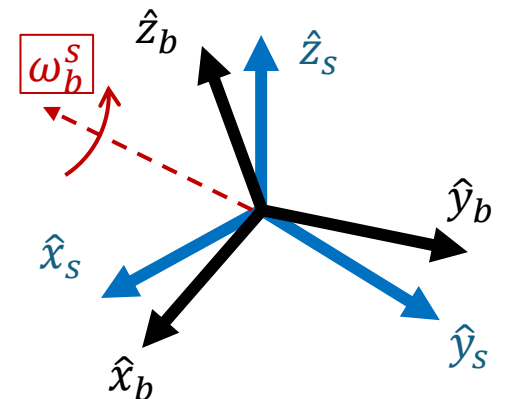
- Angular velocity expressed in  $\{*\}$ :

$$\tilde{\omega}_b^{*,S} \in T_I SO(3) =: so(3)$$

- Kinematic relation:

$$\dot{R}_b^S = R_b^S \tilde{\omega}_b^{b,S}$$

$$\dot{R}_b^S = \tilde{\omega}_b^{S,S} R_b^S$$



# Recap: Kinematic Relations

## 3. Rigid body motion :

- Configuration:

$$H_b^S \in SE(3)$$

- Rate-of-change of configuration:

$$\dot{H}_b^S \in T_{H_b^S} SE(3)$$

- Twist expressed in  $\{*\}$ :

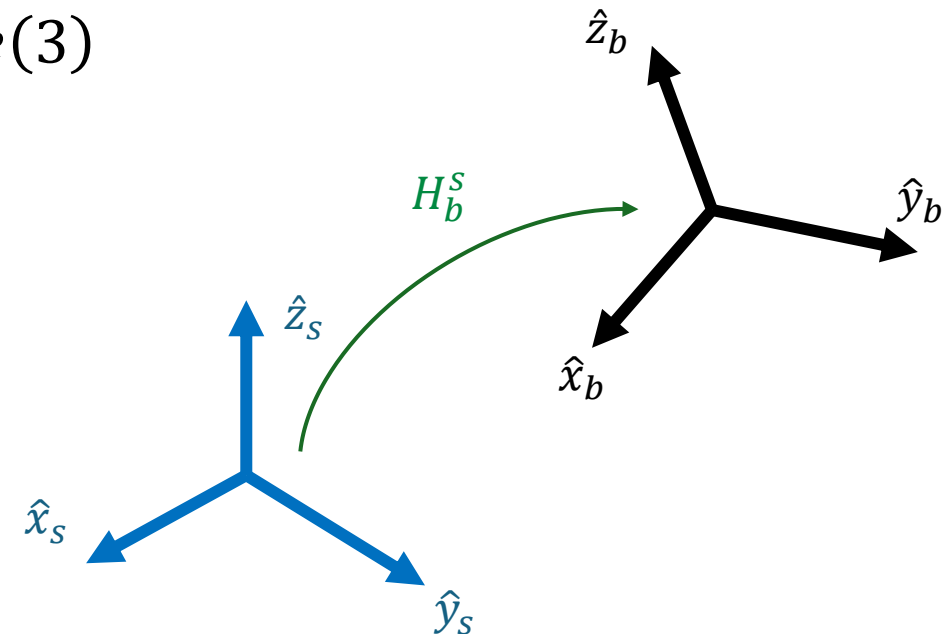
$$\tilde{\mathcal{V}}_b^{*,S} \in T_I SE(3) =: se(3)$$

- Kinematic relation:

$$\dot{H}_b^S = H_b^S \tilde{\mathcal{V}}_b^{b,S}$$

$$\dot{H}_b^S = \tilde{\mathcal{V}}_b^{S,S} H_b^S$$

$$SE(3) := SO(3) \ltimes \mathbb{R}^3$$



# Recap: Matrix form of velocities in 3D

- Angular velocity

$$\omega = \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix} \in \mathbb{R}^3$$

$$s \downarrow \quad \uparrow s^{-1}$$

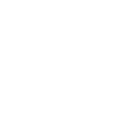
$$\tilde{\omega} = \begin{pmatrix} 0 & -\omega_z & \omega_y \\ \omega_z & 0 & -\omega_x \\ -\omega_y & \omega_x & 0 \end{pmatrix} \in so(3)$$

- Twist

$$\mathcal{V} = \begin{pmatrix} \omega \\ v \end{pmatrix} \in \mathbb{R}^6$$

$$\tilde{s} \downarrow \quad \uparrow \tilde{s}^{-1}$$

$$\tilde{\mathcal{V}} = \begin{pmatrix} \tilde{\omega} & v \\ 0_{3 \times 1} & 0 \end{pmatrix} \in se(3)$$



# Recap: Relation between spatial and body twist

- Spatial Twist

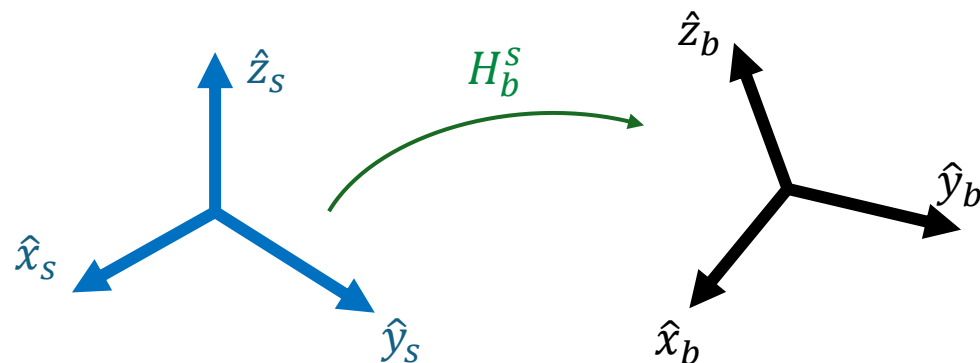
$$\mathcal{V}_b^{s,s} = \begin{pmatrix} \omega_b^{s,s} \\ \mathcal{V}_b^{s,s} \end{pmatrix} \in \mathbb{R}^6$$

- Body Twist

$$\mathcal{V}_b^{b,s} = \begin{pmatrix} \omega_b^{b,s} \\ \mathcal{V}_b^{b,s} \end{pmatrix} \in \mathbb{R}^6$$

$$\mathcal{V}_b^{s,s} = \text{Ad}_{H_b^s} \mathcal{V}_b^{b,s} \longrightarrow \text{Ad}_H = \begin{pmatrix} R & 0 \\ \tilde{\xi}R & R \end{pmatrix}$$

called the Adjoint map of  $SE(3)$



# Recap: Properties of Adjoint Map of SE(3)

- Adjoint map of SE(3)

$$\text{Ad}_H: \mathbb{R}^6 \rightarrow \mathbb{R}^6$$

$$\mathcal{V}_b^{s,s} = \text{Ad}_{H_b^s} \mathcal{V}_b^{b,s}$$

- Closed form expressions:

- Composition

$$\text{Ad}_{H_1} \text{Ad}_{H_2} = \text{Ad}_{H_1 H_2}$$

- Inverse

$$(\text{Ad}_H)^{-1} = \text{Ad}_{H^{-1}}$$

- Time derivative

$$\frac{d}{dt} \left( \text{Ad}_{H_i^k} \right) = \text{Ad}_{H_i^k} \text{ad}_{\mathcal{V}_i^{i,k}}$$

$$\text{Ad}_H = \begin{pmatrix} R & 0 \\ \tilde{\xi}R & R \end{pmatrix}$$

6 × 6  
matrix

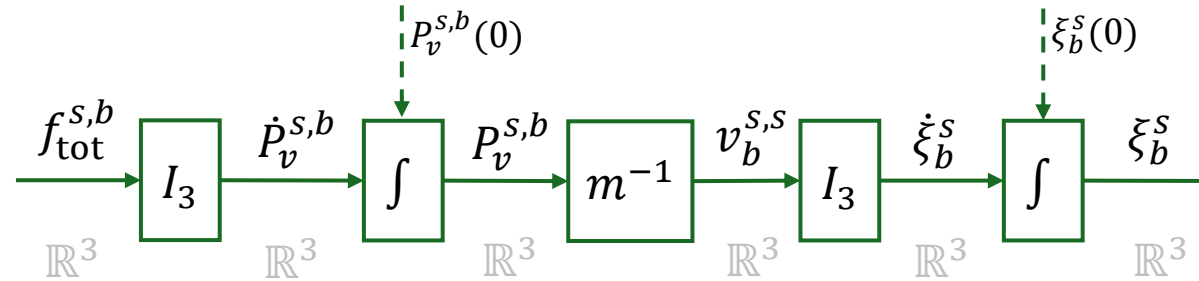
$$\text{ad}_{\mathcal{V}} = \begin{pmatrix} \tilde{\omega} & 0_{3 \times 3} \\ \tilde{v} & \tilde{\omega} \end{pmatrix}$$

6 × 6  
matrix

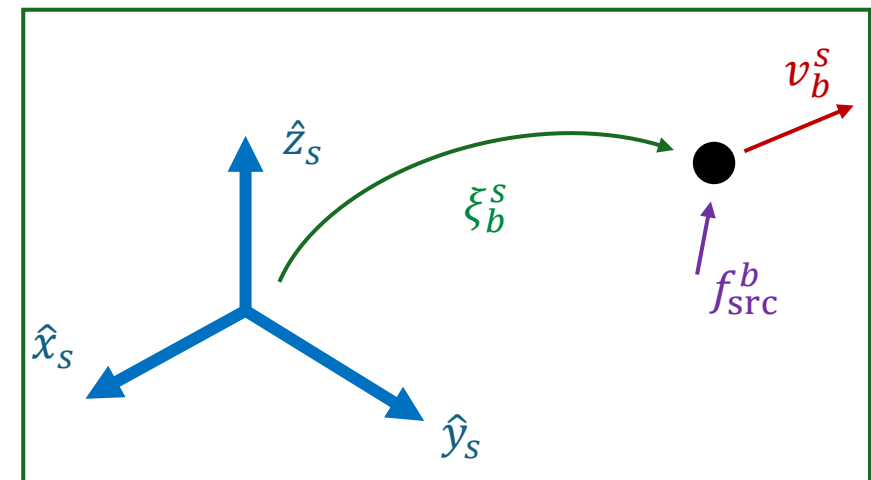
Adjoint operator of se(3)



# Recap: Point mass dynamics



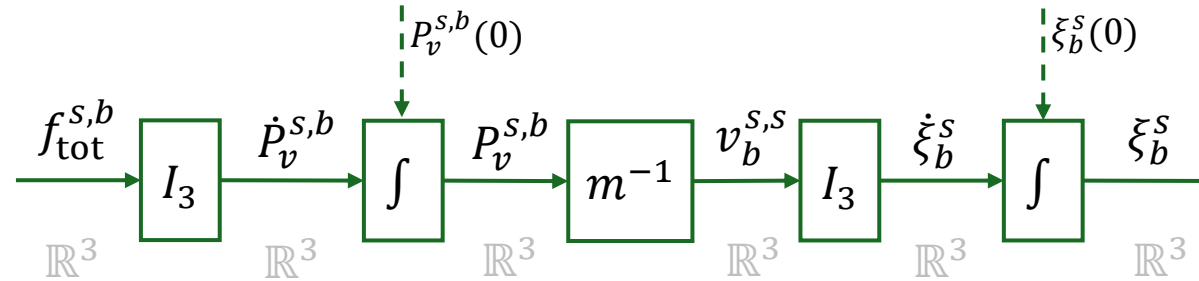
- |  |                       |
|--|-----------------------|
| • $\dot{\xi}_b^s = v_b^{s,s}$              | Kinematic relation    |
| • $\dot{P}_v^{s,b} = f_{\text{tot}}^{s,b}$ | Momentum balance      |
| • $v_b^{s,s} = m^{-1} P_v^{s,b}$           | Constitutive relation |



Forces on translating point mass



# Recap: Point mass dynamics



- $\dot{\xi}_b^s = v_b^{s,s}$
- $\dot{P}_v^{s,b} = f_{\text{tot}}^{s,b}$
- $v_b^{s,s} = m^{-1} P_v^{s,b}$

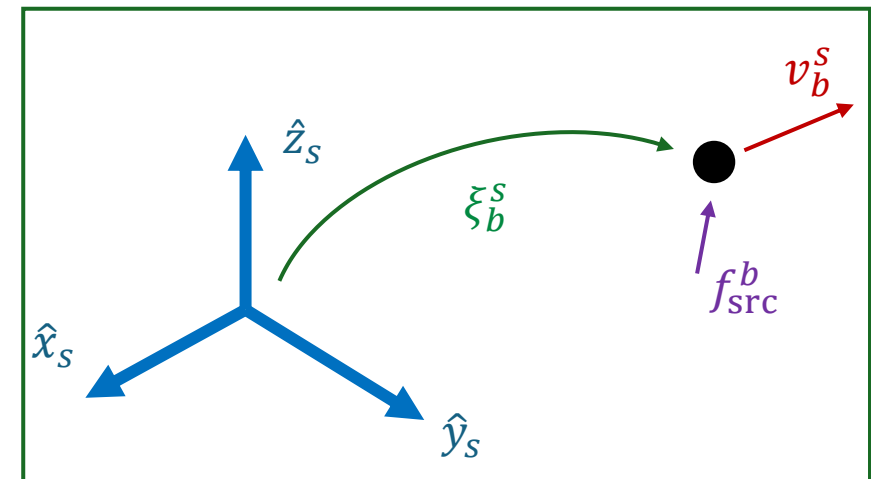
Kinematic relation

Momentum balance

Constitutive relation

- $m \ddot{\xi}_b^s = f_{\text{tot}}^{s,b}$

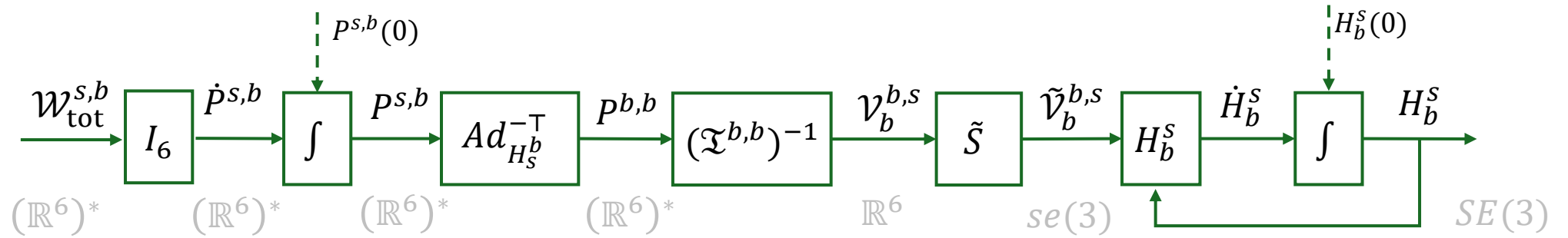
Familiar form



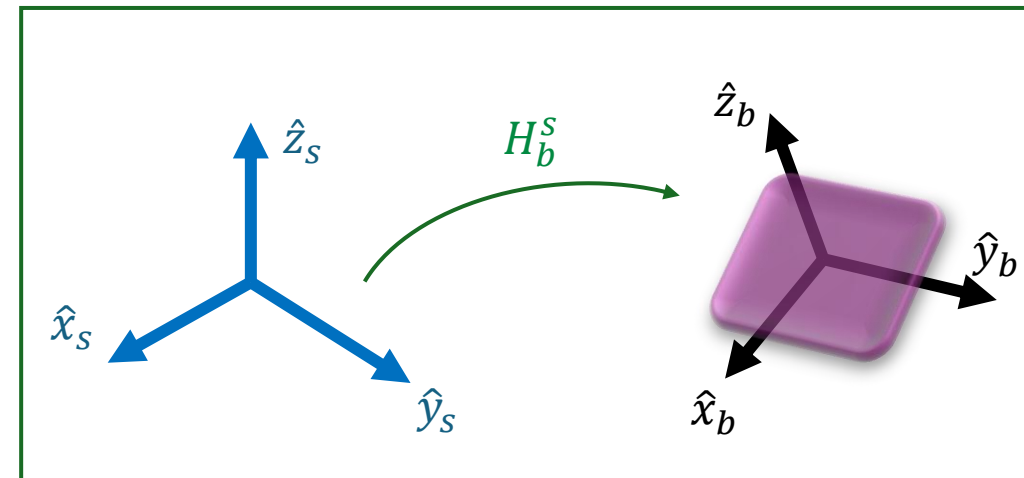
Forces on translating point mass



# Recap: Rigid body dynamics



- $\dot{H}_b^s = H_b^s \tilde{\mathcal{V}}_b^{b,s}$  Kinematic relation
- $\dot{p}^{s,b} = \mathcal{W}_{\text{tot}}^{s,b}$  Momentum balance
- $\mathcal{V}_b^{b,s} = (\mathfrak{I}^{b,b})^{-1} p^{b,b}$  Constitutive relation



$P^{*,b}$ : generalized momentum of the body expressed in  $\{*\}$   
 $\mathcal{W}_{\text{tot}}^{*,b}$ : resultant wrench on body expressed in  $\{*\}$   
 $\mathfrak{I}^{*,b}$ : generalized inertia of the body expressed in  $\{*\}$

Wrenches on moving rigid body



# Outline

- Recap last lectures
- In depth analysis



# Abstract concepts vs. numerical representations

- Abstract:

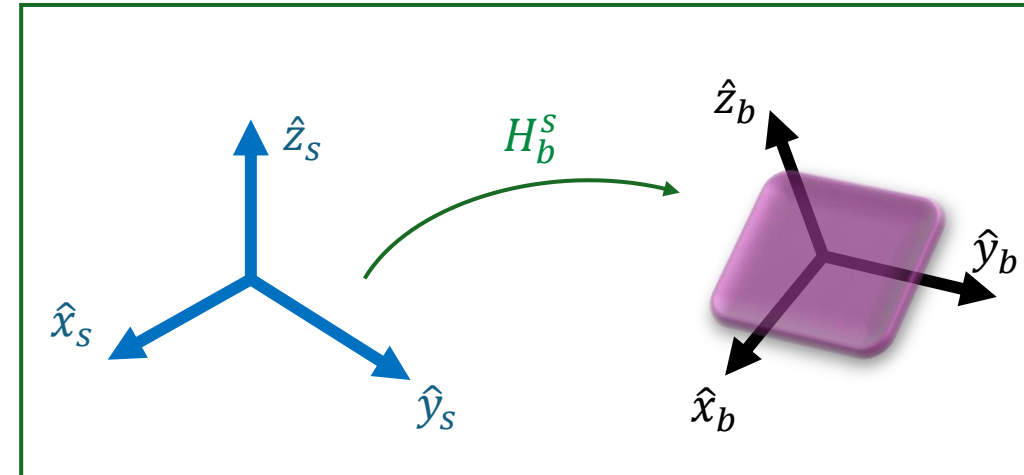
- $\mathcal{V}_b^s$ : generalized velocity of the **body** with respect to fixed world
- $P^b$ : generalized momentum of the **body**
- $\mathcal{W}_{\text{tot}}^b$ : resultant wrench on **body**
- $\mathcal{I}^b$ : generalized inertia of the **body**

$$\mathcal{V}_b^s \in \mathbb{V}, \quad P^b, \mathcal{W}_{\text{tot}}^b \in \mathbb{V}^*, \quad \mathcal{I}^b: \mathbb{V} \rightarrow \mathbb{V}^*$$

$$\begin{aligned} \bullet \quad \dot{H}_b^s &= H_b^s \tilde{\mathcal{V}}_b^{b,s} \\ \bullet \quad \dot{p}^{s,b} &= \mathcal{W}_{\text{tot}}^{s,b} \\ \bullet \quad \mathcal{V}_b^{b,s} &= (\mathcal{I}^{b,b})^{-1} p^{b,b} \end{aligned}$$

$\mathbb{V}$  as a vector space:

- $0 \in \mathbb{V}$
- $\mathcal{V}_1 + \mathcal{V}_2 \in \mathbb{V}$



Wrenches on moving rigid body



# Abstract concepts vs. numerical representations

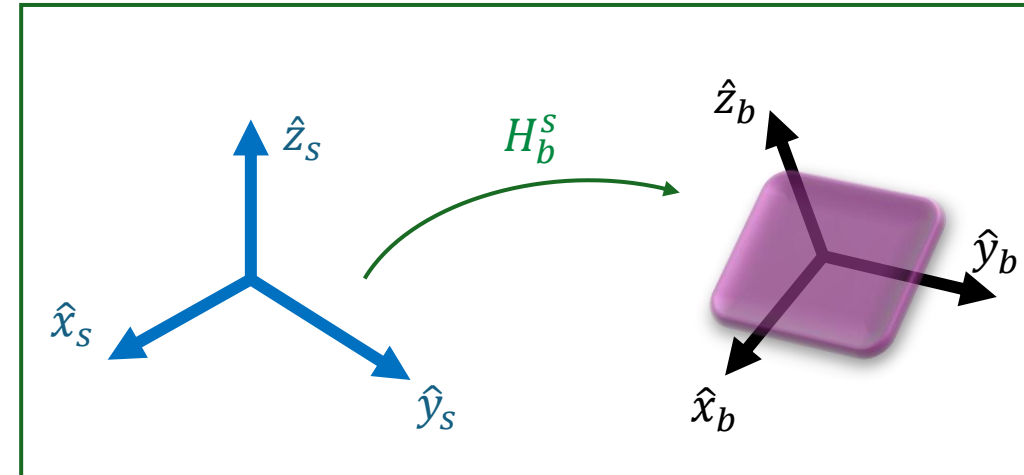
- Abstract:

- $\mathcal{V}_b^s$ : generalized velocity of the **body** with respect to fixed world
- $P^b$ : generalized momentum of the **body**
- $\mathcal{W}_{\text{tot}}^b$ : resultant wrench on **body**
- $\mathfrak{I}^b$ : generalized inertia of the **body**

$$\mathcal{V}_b^s \in \mathbb{V}, \quad P^b, \mathcal{W}_{\text{tot}}^b \in \mathbb{V}^*, \quad \mathfrak{I}^b: \mathbb{V} \rightarrow \mathbb{V}^*$$

- $P^b := \mathfrak{I}^b(\mathcal{V}_b^s)$ : definition of the momentum of the **body**
- $\langle P^b | \mathcal{V}_b^s \rangle$ : twice the kinetic energy of the **body**
- $\langle \mathcal{W}_{\text{tot}}^b | \mathcal{V}_b^s \rangle$ : the mechanical power due to wrench

$$\begin{aligned} \bullet \quad \dot{H}_b^s &= H_b^s \tilde{\mathcal{V}}_b^{b,s} \\ \bullet \quad \dot{P}^{s,b} &= \mathcal{W}_{\text{tot}}^{s,b} \\ \bullet \quad \mathcal{V}_b^{b,s} &= (\mathfrak{I}^{b,b})^{-1} P^{b,b} \end{aligned}$$



$$\langle \cdot | \cdot \rangle: \mathbb{V}^* \times \mathbb{V} \rightarrow \mathbb{R}$$

Duality product/pairing

Wrenches on moving rigid body



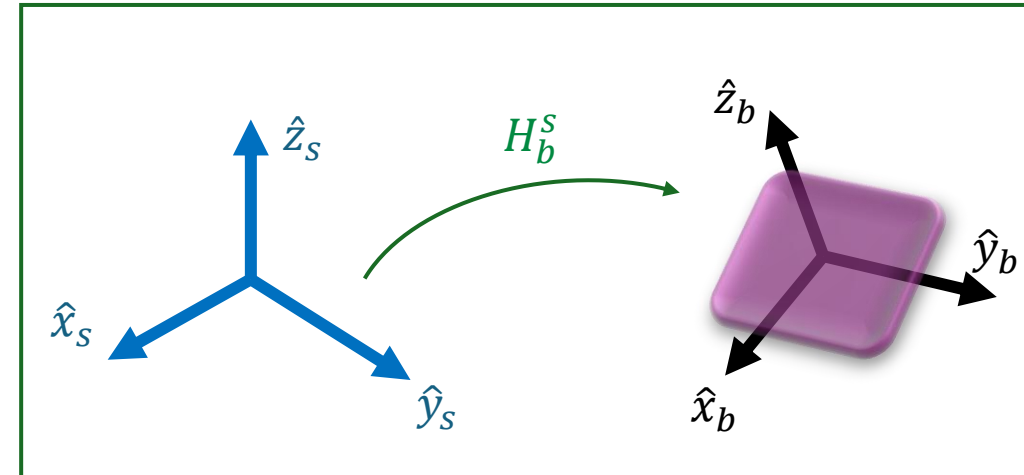
# Abstract concepts vs. numerical representations

- Components in  $\{m\}$ :
  - $\mathcal{V}_b^{m,s} := [\mathcal{V}_b^s]_m \in \mathbb{R}^6$
  - $P^{m,b} := [P^b]_m \in (\mathbb{R}^6)^*$
  - $\mathcal{W}_{\text{tot}}^{m,b} := [\mathcal{W}_{\text{tot}}^b]_m \in (\mathbb{R}^6)^*$
  - $\mathfrak{T}^{m,b} := [\mathfrak{T}^b]_m: \mathbb{R}^6 \rightarrow (\mathbb{R}^6)^*$

- Change of basis from  $\{m\}$  to  $\{k\}$

- $\mathcal{V}_b^{k,s} = \text{Ad}_{H_m^k} \mathcal{V}_b^{m,s}$
- $P^{k,b} = \text{Ad}_{H_m^k}^{-\top} P^{m,b}$
- $\mathcal{W}_{\text{tot}}^{k,b} = \text{Ad}_{H_m^k}^{-\top} \mathcal{W}_{\text{tot}}^{m,b}$
- $\mathfrak{T}^{k,b} = \text{Ad}_{H_m^k}^{-\top} \mathfrak{T}^{m,b} \text{Ad}_{H_m^k}^{-1}$

- $\dot{H}_b^s = H_b^s \tilde{\mathcal{V}}_b^{b,s}$
- $\dot{P}^{s,b} = \mathcal{W}_{\text{tot}}^{s,b}$
- $\mathcal{V}_b^{b,s} = (\mathfrak{T}^{b,b})^{-1} P^{b,b}$



Wrenches on moving rigid body



# Abstract concepts vs. numerical representations

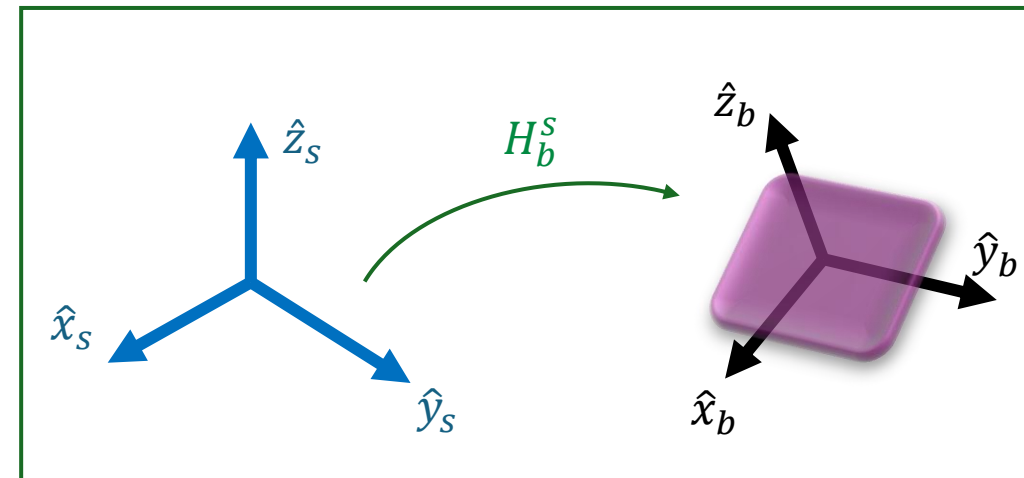
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- $\mathfrak{T}^{m,b} := [\mathfrak{T}^b]_m: \mathbb{R}^6 \rightarrow (\mathbb{R}^6)^*$

- Change of basis from  $\{m\}$  to  $\{k\}$

- $\mathcal{V}_b^{k,s} = \text{Ad}_{H_m^k} \mathcal{V}_b^{m,s} = \text{Ad}_{H_k^m}^{-1} \mathcal{V}_b^{m,s}$
- $P^{k,b} = \text{Ad}_{H_m^k}^{-\top} P^{m,b} = \text{Ad}_{H_k^m}^{\top} P^{m,b}$
- $\mathcal{W}_{\text{tot}}^{k,b} = \text{Ad}_{H_m^k}^{-\top} \mathcal{W}_{\text{tot}}^{m,b} = \text{Ad}_{H_k^m}^{\top} \mathcal{W}_{\text{tot}}^{m,b}$
- $\mathfrak{T}^{k,b} = \text{Ad}_{H_m^k}^{-\top} \mathfrak{T}^{m,b} \text{Ad}_{H_m^k}^{-1} = \text{Ad}_{H_k^m}^{\top} \mathfrak{T}^{m,b} \text{Ad}_{H_k^m}$

- $\dot{H}_b^s = H_b^s \tilde{\mathcal{V}}_b^{b,s}$
- $\dot{P}^{s,b} = \mathcal{W}_{\text{tot}}^{s,b}$
- $\mathcal{V}_b^{b,s} = (\mathfrak{T}^{b,b})^{-1} P^{b,b}$



Wrenches on moving rigid body



# Generalized Inertia

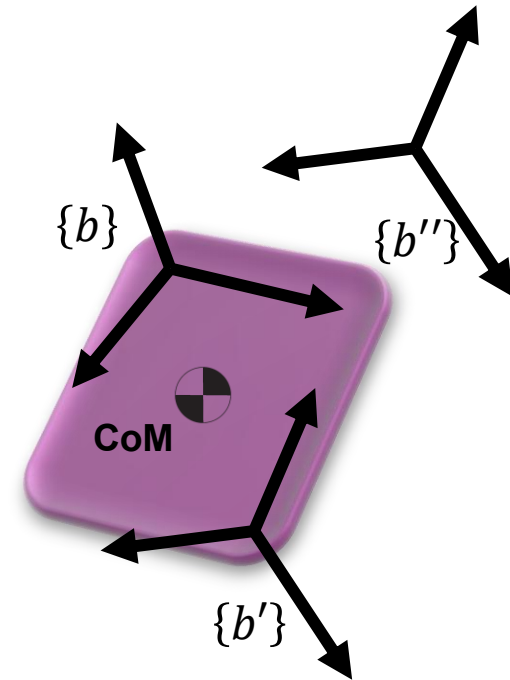
- $\mathcal{I}^{k,b}$  is always symmetric & positive definite
- $\mathcal{I}^{k,b}$  is constant if  $\{k\}$  is **any** body-fixed frame

$$\mathcal{I}^{k,b} = \begin{pmatrix} J^{k,b} & m \tilde{\xi}_{\text{cm}}^k \\ -m \tilde{\xi}_{\text{cm}}^k & m I_3 \end{pmatrix}$$

- $\mathcal{I}^{k,b}$  is block-diagonal if  $\{k\}$  is at CoM

$$\mathcal{I}^{k,b} = \begin{pmatrix} J^{k,b} & 0_{3 \times 3} \\ 0_{3 \times 3} & m I_3 \end{pmatrix}$$

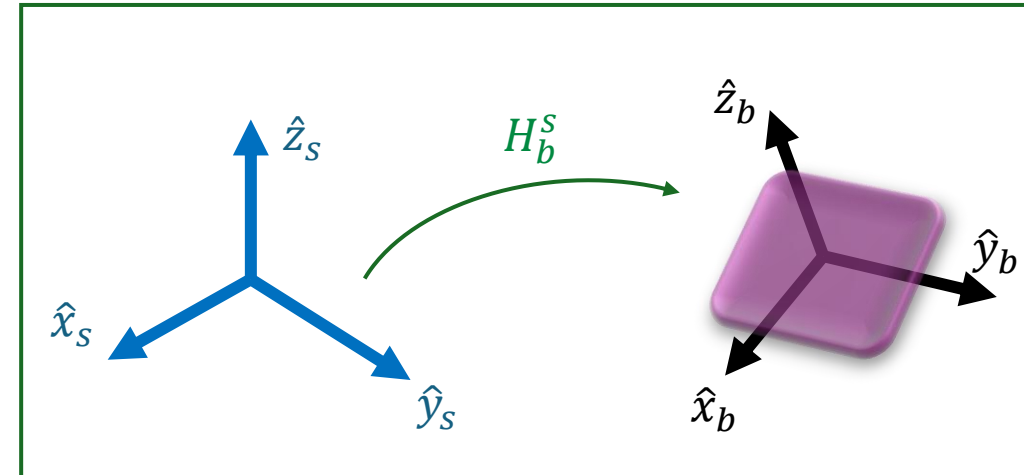
$$J^{k,b} = \begin{pmatrix} J_{xx} & J_{xy} & J_{xz} \\ J_{xy} & J_{yy} & J_{yz} \\ J_{xz} & J_{yz} & J_{zz} \end{pmatrix} : \text{moment of inertia matrix}$$



# Up Next !

- How does the momentum balance equation look like in terms of  $P^{b,b}$  ?
- Can we split rotational and translational dynamics ?
- How is this related to more common forms in the literature ?
- What is the twist ?

$$\begin{aligned} \bullet \quad \dot{H}_b^s &= H_b^s \tilde{\mathcal{V}}_b^{b,s} \\ \bullet \quad \dot{P}^{s,b} &= \mathcal{W}_{\text{tot}}^{s,b} \\ \bullet \quad \mathcal{V}_b^{b,s} &= (\mathcal{I}^{b,b})^{-1} p^{b,b} \end{aligned}$$



Wrenches on moving rigid body

