

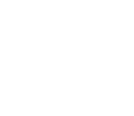
SCE 594: Special Topics in Intelligent Automation & Robotics

Lecture 22: Stabilization Control on $SO(3)$ II



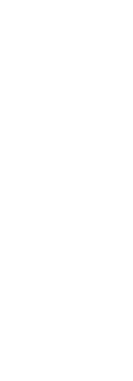
Outline

- Recap last lectures
- Stabilization Control on $SO(3)$...



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Recap: Energy-balancing interpretation of PD Control

- The PD controller

$$u = u_p + u_d$$

can be interpreted as a sum of an energy-shaping term u_p and a damping injection term u_d .

- For a chosen locally positive definite function $\Psi(\xi)$ designed such that ξ_d is a minimum, one has that $u_p = -\nabla\Psi(\xi)$ which yields $\dot{V}(x) = v^\top u_d$.



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- For a chosen locally positive definite function $\Psi(\xi)$ designed such that ξ_d is a minimum, one has that $u_p = -\nabla\Psi(\xi)$ which yields $\dot{V}(x) = v^\top u_d$.
- Choosing $u_d = \gamma(v)$ to inject damping such that $\dot{V}(x) \leq 0$, one has with La Salle's invariance principle that $x_d = (\xi_d, 0)$ is locally asymptotically stable.
- If $\Psi(\xi)$ has ξ_d to be a **global minimum**, then $x_d = (\xi_d, 0)$ is **globally asymptotically stable**.



Recap: Rigid Body Rotation Dynamics

- The equations of a rotating rigid body with control torques τ are:

- $x = (R, p) \in SO(3) \times \mathbb{R}^3$

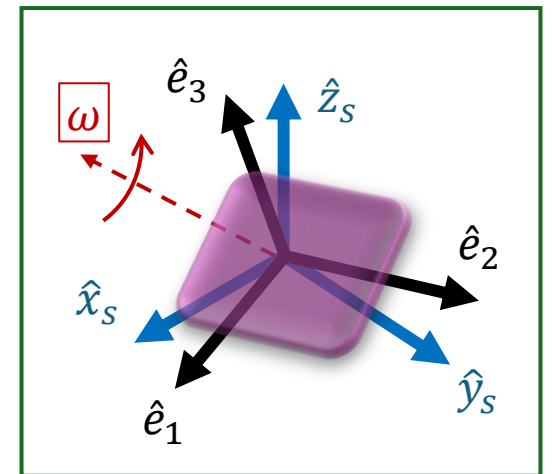
- $\begin{pmatrix} \dot{R} \\ \dot{p} \end{pmatrix} = \begin{pmatrix} \beta_R(\omega) \\ \tilde{p} \omega \end{pmatrix} + \begin{pmatrix} 0 \\ I_3 \end{pmatrix} \tau$

with $\omega = J^{-1}p$

and $\beta_R: \mathbb{R}^3 \rightarrow T_R SO(3), \quad \omega \mapsto \beta_R(\omega) := R \tilde{\omega}$

- Total energy of the system

- $H(x) = H(p) = \frac{1}{2} p^\top J^{-1} p$



Recap: Stabilization Control on SO(3)

- We aim to design a **Geometric** PD controller

$$\tau = \tau_p + \tau_d$$

such that $x_d = (R_d, 0)$ is an asymptotically stable equilibrium point of the closed loop system

$$\begin{pmatrix} \dot{R} \\ \dot{p} \end{pmatrix} = \begin{pmatrix} \beta_R(\omega) \\ \tilde{p} \omega + \tau_p + \tau_d \end{pmatrix}.$$



Recap: Stabilization Control on $SO(3)$

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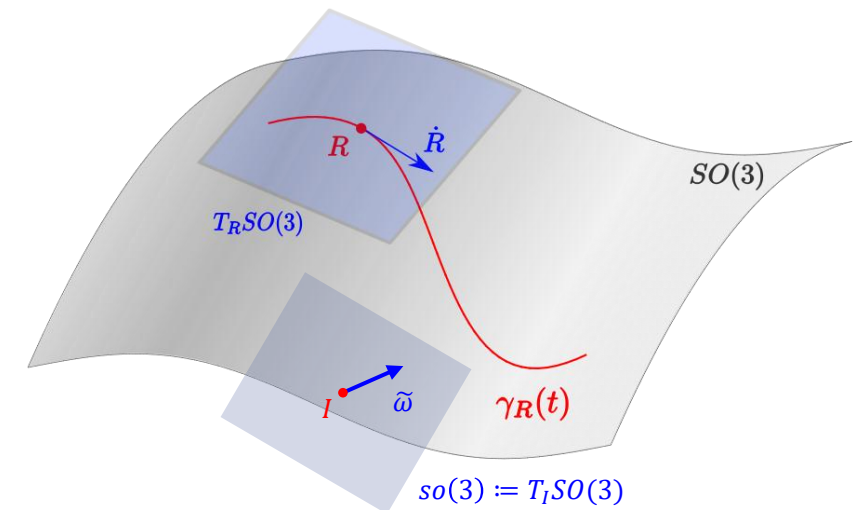
gradient of some pos.def. function $\Psi(R)$
on $SO(3)$ with a minimum at $R = R_d$.

Inject damping such that time derivative
of $V(x) = H(p) + \Psi(R)$ is neg. def.



Recap: Geometric structure of $SO(3)$

- The geometric nature of $SO(3)$ will be reflected in
 1. How to compute the error between $R, R_d \in SO(3)$?
 2. How to design $\Psi(R)$ to be positive definite ?
 3. How to compute $d\Psi(R) \in T_R^*SO(3)$?
 4. How to convert $d\Psi(R)$ to the proportional torque $\tau_p \in \mathbb{R}^3$?
 5. How to design the derivative torque $\tau_d \in \mathbb{R}^3$?



Recap: Stabilization Control on $SO(3)$

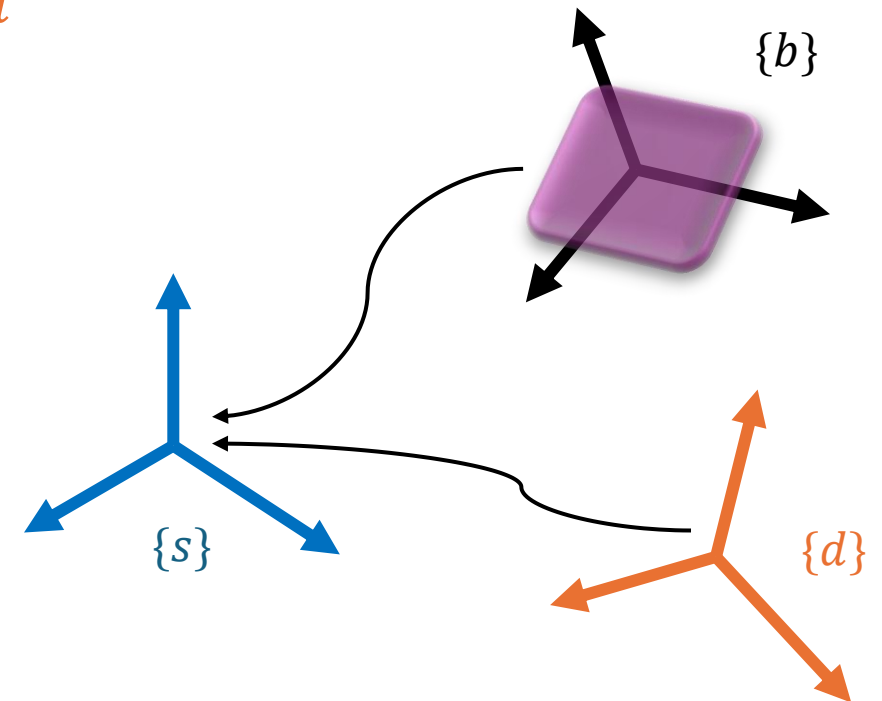
1. How to compute the error between $R, R_d \in SO(3)$?

- Error between actual and desired

$$R_e := R_d^T R \in SO(3)$$

- We have that

$$R_e \rightarrow I \text{ as } R \rightarrow R_d$$



Recap: Stabilization Control on $SO(3)$

2. How to design $\Psi(R)$ to be positive definite ?

- One choice for $\Psi: SO(3) \rightarrow \mathbb{R}$ is

$$\Psi(R) := \frac{1}{2} \operatorname{tr}(I - R_d^T R)$$

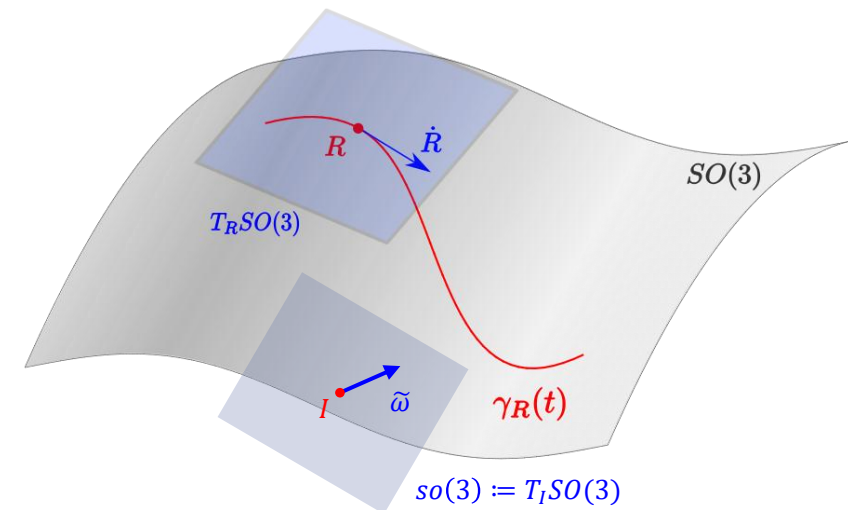
- $\Psi(R_d) = 0$
- We have shown before that it can be written as

$$\Psi(R) = 1 - \cos \theta$$

for some $\theta \in (-\pi, \pi]$.

- Therefore,

$$0 \leq \Psi(R) \leq 2$$



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PD Control on $SO(3)$

3. How to compute $d\Psi(R) \in T_R^*SO(3)$?

- The differential of $\Psi: SO(3) \rightarrow \mathbb{R}$ is defined as the unique covector $d\Psi(R) \in T_R^*SO(3)$ that satisfies

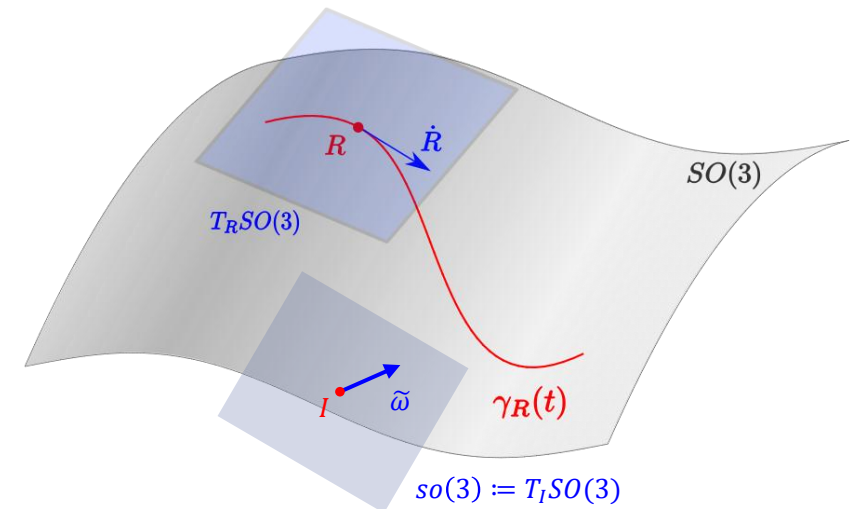
$$\langle d\Psi(R) | \delta R \rangle_{T_R SO(3)} = \left. \frac{d}{d\epsilon} \right|_{\epsilon=0} \Psi(R_\epsilon), \quad \forall \delta R \in T_R SO(3)$$

where R_ϵ is a curve on $SO(3)$ that satisfies

$$R_\epsilon|_{\epsilon=0} = R, \quad \left. \frac{d}{d\epsilon} \right|_{\epsilon=0} R_\epsilon = \delta R$$

The pairing between any covector in $T_R^*SO(3)$ and vector $T_R SO(3)$ is:

$$\langle \Gamma | \delta R \rangle_{T_R SO(3)} := \text{tr}(\Gamma \delta R)$$



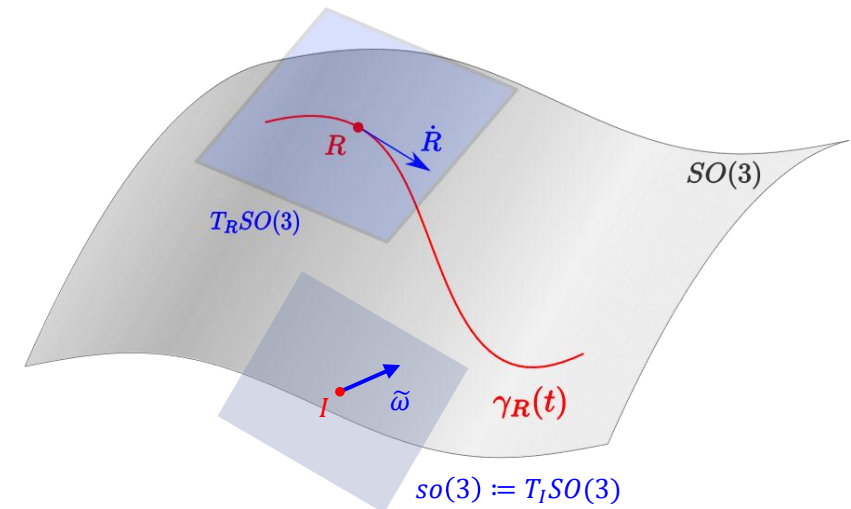
PD Control on $SO(3)$

3. How to compute $d\Psi(R) \in T_R^*SO(3)$?

- $\Psi(R_\epsilon) = \frac{1}{2} \text{tr} (I - R_d^\top R_\epsilon)$
- $\frac{d}{d\epsilon} \Big|_{\epsilon=0} \Psi(R_\epsilon) = -\frac{1}{2} \text{tr} \left(R_d^\top \frac{d}{d\epsilon} \Big|_{\epsilon=0} R_\epsilon \right) = -\frac{1}{2} \text{tr} (R_d^\top \delta R)$
- $\langle d\Psi(R) | \delta R \rangle_{T_R SO(3)} = \text{tr} (d\Psi(R) \delta R)$

- By comparing both sides, we have that

$$d\Psi(R) = -\frac{1}{2} R_d^\top$$

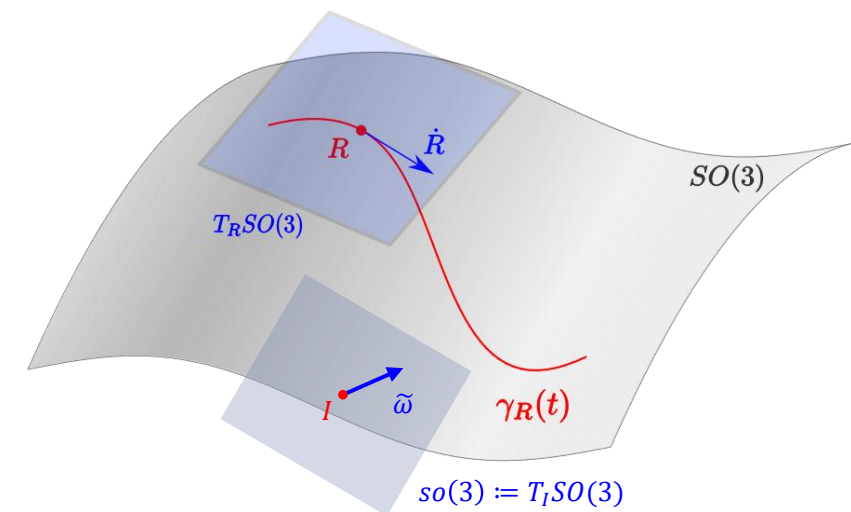


PD Control on $SO(3)$

4. How to convert $d\Psi(R)$ to the proportional torque $\tau_p \in \mathbb{R}^3$?

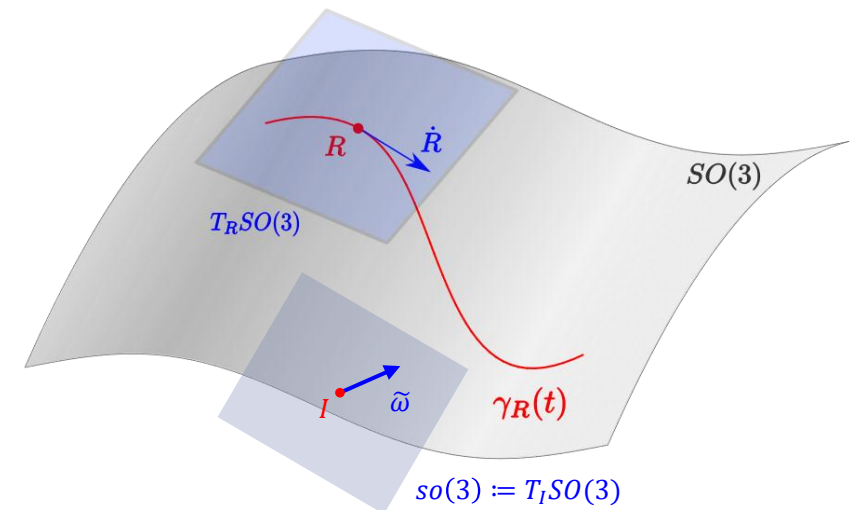
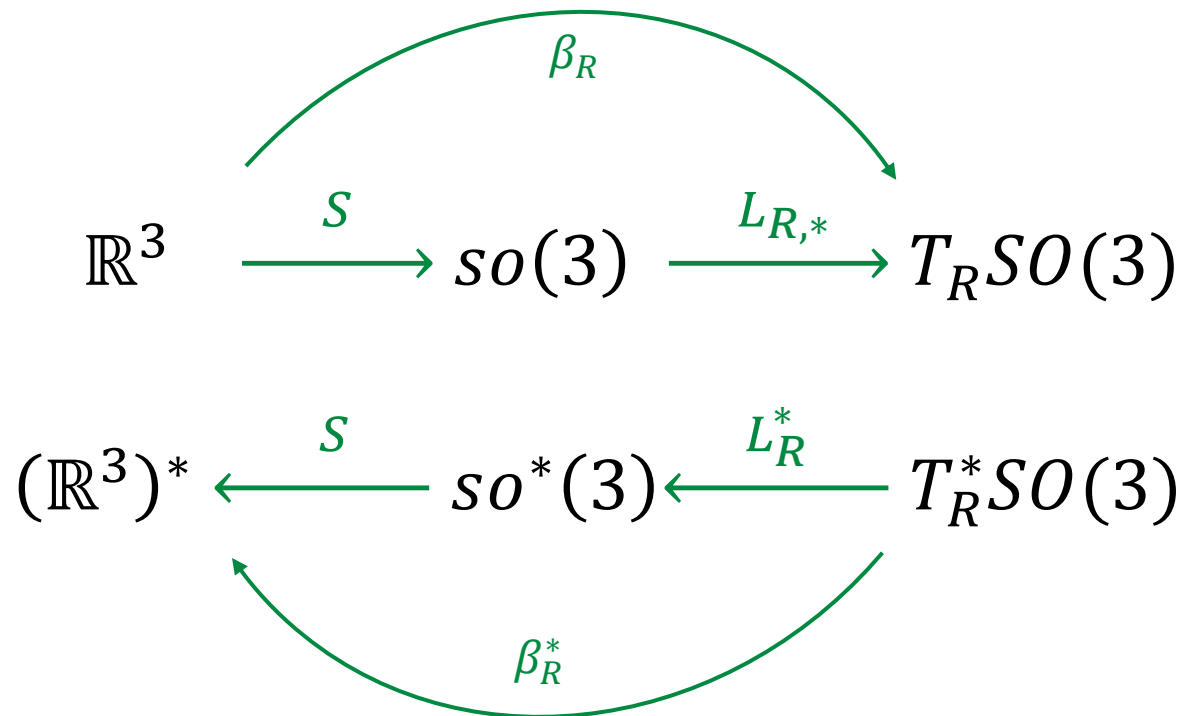
- We use the duality of linear maps !
- Let $A: U \rightarrow V$ be a linear map between the vector spaces U and V . Then its dual map $A^*: V^* \rightarrow U^*$ is defined (implicitly) by:

$$\langle A^*(\alpha) | u \rangle_U = \langle \alpha | A(u) \rangle_V, \quad \forall u \in U, \alpha \in V^*$$



PD Control on $SO(3)$

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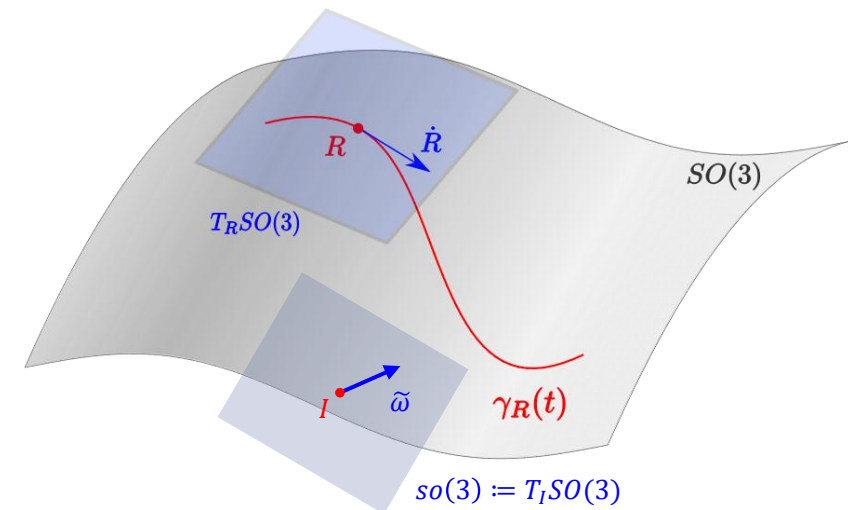


PD Control on $SO(3)$

4. How to convert $d\Psi(R)$ to the proportional torque $\tau_p \in \mathbb{R}^3$?

- $d\Psi(R) = -\frac{1}{2}R_d^\top \in T_R^*SO(3)$
- $\tilde{e}_R := L_R^*(d\Psi(R)) = \text{sk}(R_d^\top R) = \frac{1}{2}(R_d^\top R - R^\top R_d) \in \mathfrak{so}^*(3)$
- $e_R := \beta_R^*(d\Psi(R)) = S^{-1}(\tilde{e}_R) = \frac{1}{2}S^{-1}(R_d^\top R - R^\top R_d) \in (\mathbb{R}^3)^*$
- The proportional torque is then designed as

$$\tau_p := -k_p e_R$$



Proportional gain: $k_p \in \mathbb{R}_+$



PD Control on $SO(3)$

5. How to design the derivative torque $\tau_d \in \mathbb{R}^3$?

- Let's conduct a Lyapunov analysis for the closed loop system

$$\begin{pmatrix} \dot{R} \\ \dot{p} \end{pmatrix} = \begin{pmatrix} \beta_R(\omega) \\ \tilde{p} \omega + \tau_p + \tau_d \end{pmatrix} = \begin{pmatrix} \beta_R(\omega) \\ \tilde{p} \omega - k_p \beta_R^*(d\Psi(R)) + \tau_d \end{pmatrix}$$

